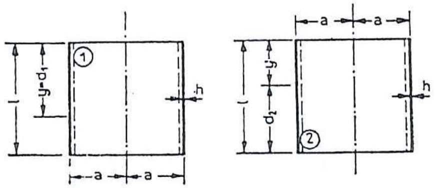


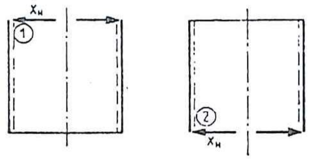
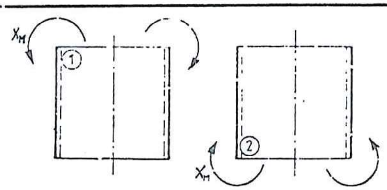
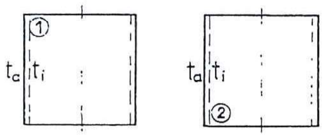
$h = \text{konst.};\ k = \sqrt{\frac{\sqrt{3(1-\mu^2)}}{ah}};\ \eta = \frac{y}{l};\ d_1 = y;\ d_2 = l - y;\ n = 1 \div 2;$

$\eta_1 = e^{-kd_n}\cos kd_n;\ \eta_2 = e^{-kd_n}\sin kd_n;\ \eta_3 = \eta_1 + \eta_2;\ \eta_4 = \eta_1 - \eta_2.$

bei $d_n = 0$ wird : $\eta_1 = 1$; $\eta_2 = 0$; $\eta_3 = 1$; $\eta_4 = 1$

Die Formeln der Tabelle sind gültig für: $kl > 6$.

5.1 Zylinderschale mit gleichbleibender Wanddicke

Zylinderschale und Belastungsbild	$X_H = H_n$	X_M	N_φ	Q_y	M_y	Δr	ϑ
	X_H	0	$2akX_H\eta_1$	$\pm X_H\eta_4$	$\frac{1}{k}X_H\eta_2$	$\frac{2a^2k}{Eh}X_H\eta_1$	$\pm\frac{2a^2k^2}{Eh}X_H\eta_3$
	0	X_M	$2ak^2X_M\eta_4$	$\mp 2kX_M\eta_2$	$X_M\eta_3$	$\frac{2a^2k^2}{Eh}X_M\eta_4$	$\pm\frac{4a^2k^3}{Eh}X_M\eta_1$
	0	$\frac{\alpha_t\Delta t\,Eh^2}{6(1-\mu)}$	$\frac{ak^2\alpha_t\Delta t\,Eh^2}{3(1-\mu)}\eta_4$	$\mp\frac{k\alpha_t\Delta t\,Eh^2}{3(1-\mu)}\eta_2$	$\frac{\alpha_t\Delta t\,Eh^2}{6(1-\mu)}(\eta_3 - 1)$	$a\alpha_t\left[\frac{ak^2\Delta t\,h}{3(1-\mu)}\eta_4 + t\right]$	$\pm\frac{2a^2k^3\alpha_t\Delta t\,h}{3(1-\mu)}\eta_1$

$t = \frac{t_i + t_a}{2},\ \Delta t = \frac{t_i - t_a}{2};$

$M_\varphi = \frac{\alpha_t\Delta t\,Eh^2}{6(1-\mu)}(\mu\eta_3 - 1)$